

Analysis of Powertrain Mounting System

COMSOL
CONFERENCE
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Rise.



Content

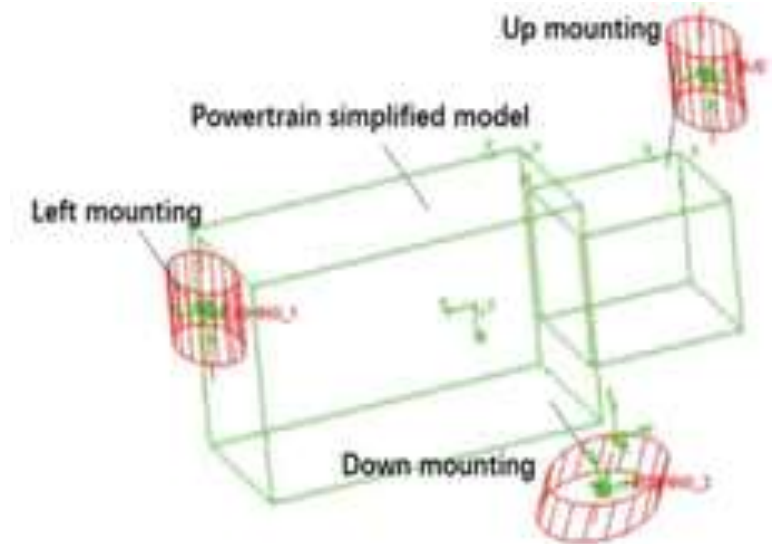
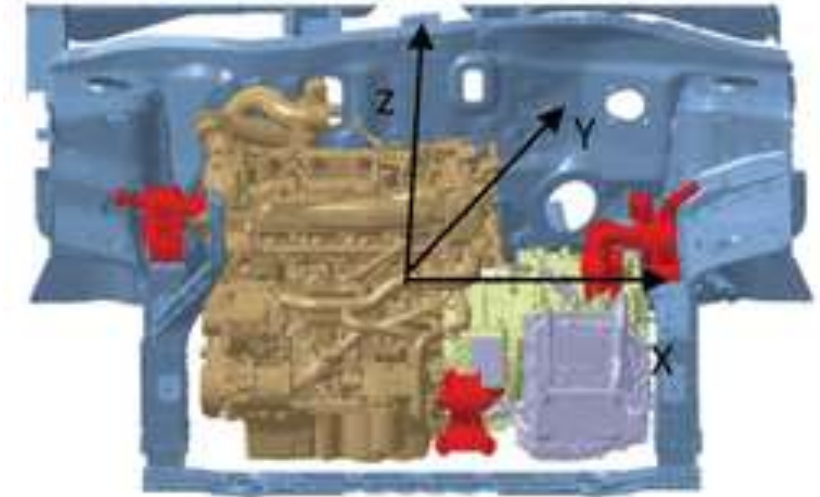
- Simulation Objective
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- Summary

Simulation Objective

- Determines the optimum engine mount configuration from the minimum set of data available at the early vehicle concept stage
- Helps to design mounts close to their optimum position and to make more realistic full vehicle models for conceptual designs
- Used to calculate the eigenfrequencies, rotations and displacements of engine mounts

Powertrain Mount

- Powertrain is having considerable mass and inertia.
- Primary Excitation is through the Engine excitation and Road excitation.
- Major role in Vehicle NVH and passenger Ride & Comfort
- Modal frequency and kinetic Energy of decoupled system



6-DOF Model of Powertrain on Mounts

Equation of Motion $[M][\ddot{x}] + [C][\dot{x}] + [K][x] = 0$

Eigen Value

$$f_i = \frac{\sqrt{D_i}}{2\pi}$$

Natural Frequency

Equation of Motion without Damping $[M][\ddot{x}] + [K][x] = 0$

$$[K] - \omega_i^2[M] = 0$$

Eigen Vector

$$[E_i] = [[V_i]^T[M][V_i]][n]^{-1}$$

Energy

$$[M] = \begin{bmatrix} m & 0 & 0 & 0 & 0 & 0 \\ 0 & m & 0 & 0 & 0 & 0 \\ 0 & 0 & m & 0 & 0 & 0 \\ 0 & 0 & 0 & I_{xx} & -I_{xy} & -I_{xz} \\ 0 & 0 & 0 & -I_{yx} & I_{yy} & -I_{yz} \\ 0 & 0 & 0 & -I_{zx} & -I_{zy} & I_{zz} \end{bmatrix}$$

$$[K] = \sum_{i=1}^n [B_i]^T [T_i]^T [k_i] [T_i] [B_i]$$

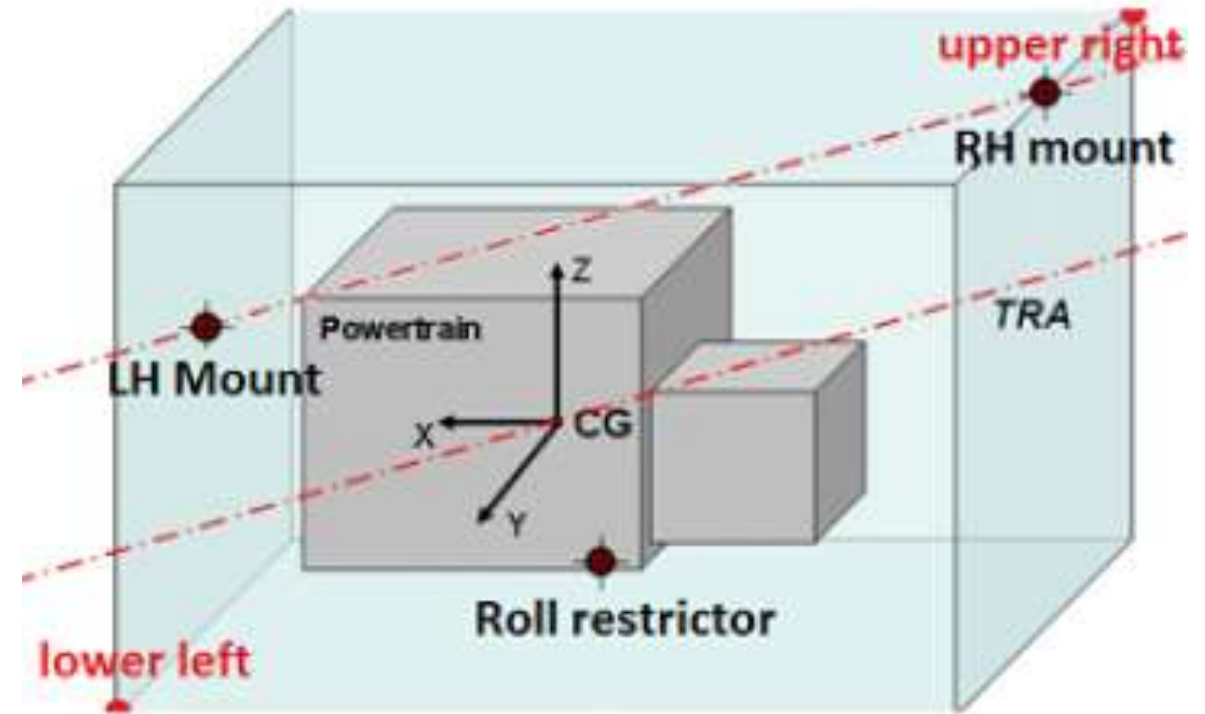
$$[B_i] = \begin{bmatrix} 1 & 0 & 0 & 0 & z_i & -y_i \\ 0 & 1 & 0 & -z_i & 0 & x_i \\ 0 & 0 & 1 & y_i & -x_i & 0 \end{bmatrix}$$

$$[T_i] = \begin{bmatrix} \cos(X'X) & \cos(Y'X) & \cos(Z'X) \\ \cos(X'Y) & \cos(Y'Y) & \cos(Z'Y) \\ \cos(X'Z) & \cos(Y'Z) & \cos(Z'Z) \end{bmatrix}$$

Analysis

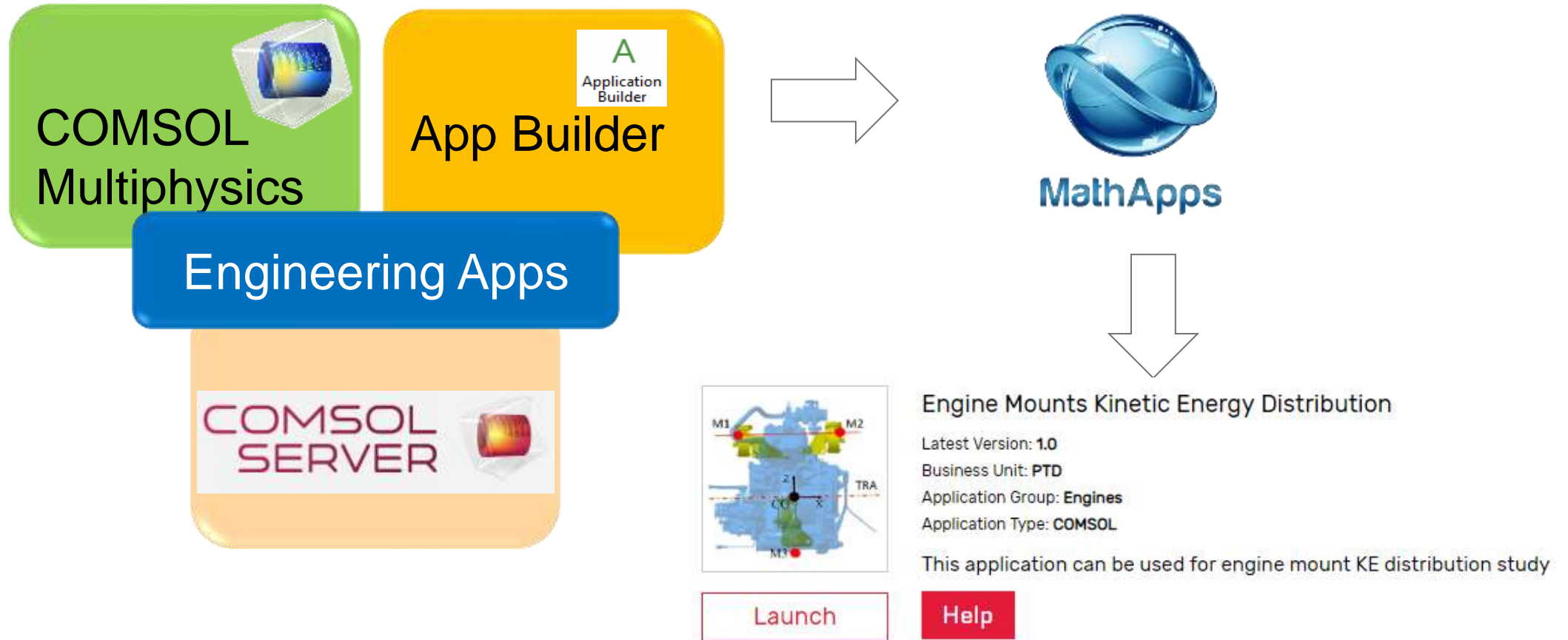
- Eigen Frequency Study
 - Eigen Frequency
 - Modal Shapes

- Stationary Study
 - Packaging
 - Mount Loads



Web Based Mathapps GUI

- Deployment of the application on our web based portal across our organization “Mathapps”



COMSOL Application



INPUT

RESULT

Please choose the desired Load case and close the Dialogue Box

- | | | | |
|----|---|----|--|
| 1 | Static Design Position (Under PT Self Weight) | 12 | Max reverse engine torque & rearward acceleration RWD(-0.5g) |
| 2 | Max forward Engine Torque | 13 | Max reverse engine torque & rearward acceleration AWD(-0.6g) |
| 3 | Max reverse engine torque | 14 | 8 KPH front bumper (-11g) |
| 4 | Max Forward Engine Torque and forward accelerat | 15 | 8 KPH rear bumper (+11g) |
| 5 | Max Forward Engine Torque and forward accelerat | 16 | Vertical up loading (+5g) |
| 6 | Max Forward Engine Torque and forward accelerat | 17 | Vertical down loading(-5g) |
| 7 | Max forward engine torque & +1g left cornering | 18 | Lateral left loading |
| 8 | Max forward engine torque & -1g right cornering | 19 | Lateral right loading |
| 9 | Max forward engine torque & -2g bump | 20 | Vertical +5g up & -3g lateral left loading |
| 10 | Max forward engine torque & +2g rebound | 21 | Vertical +5g up & +3g lateral right loading |
| 11 | Max reverse engine torque & rearward accelerati | 22 | Vertical -5g down & -3g lateral left loading |

Load Case
 Maximum engine torque
 First gear ratio:
 Reverse gear ratio:
 Multiplication factor:
 Stall torque ratio:
 Acceleration due to gra
 Final drive ratio:
 Fx
 Fy

Results

Eigenfrequency Study Stationary Study

Kinetic Energy Distribution (%)

Eigenfrequency (Hz)	X	Y	Z	X-Rot	Y-Rot	Z-Rot
4.979	99.999	0	0	0	0.001	0
8.009	0.205	0.054	0	99.723	0	0.017
11	0	0	100	0	0	0
16.18	0.378	0.027	0	0.166	99.151	0.278
17.808	0	99.987	0	0	0	0.012
20.937	0.007	5.31	0	1.195	0.095	93.392

Mode Shapes

Eigenfrequency (Hz): 4.9794 Scale factor: 0.002731

Eigenfrequency=4.9794 Hz Surface: Total displacement (mm)

Correlation: Eigen Frequency

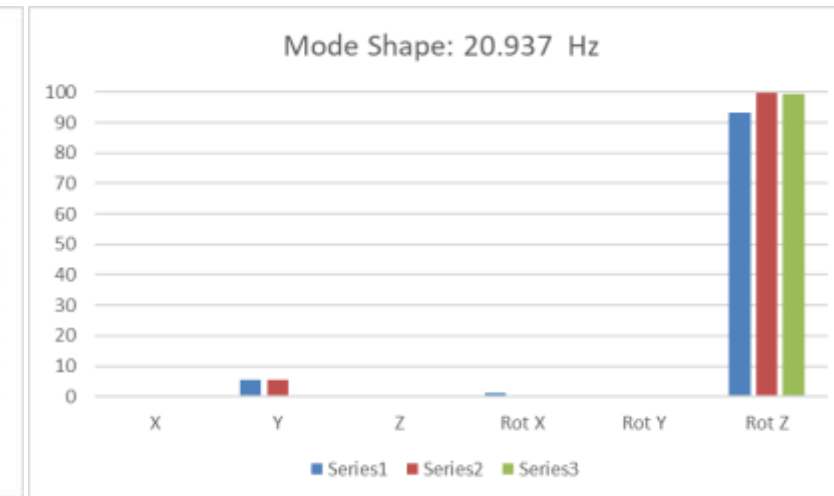
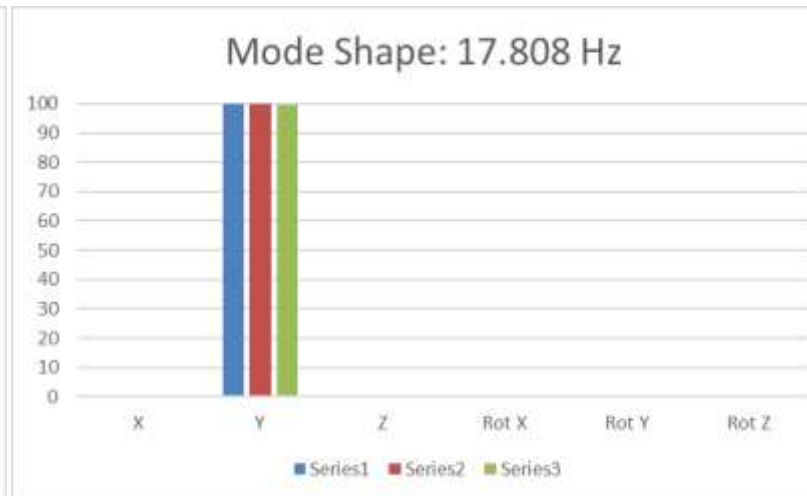
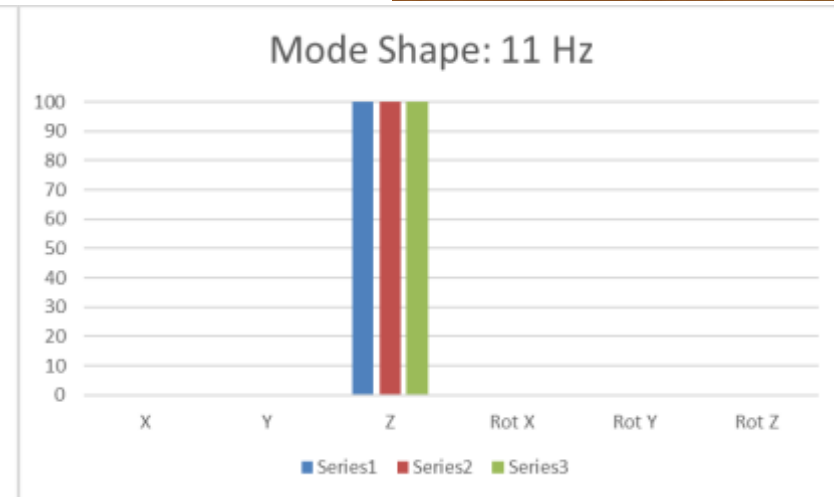
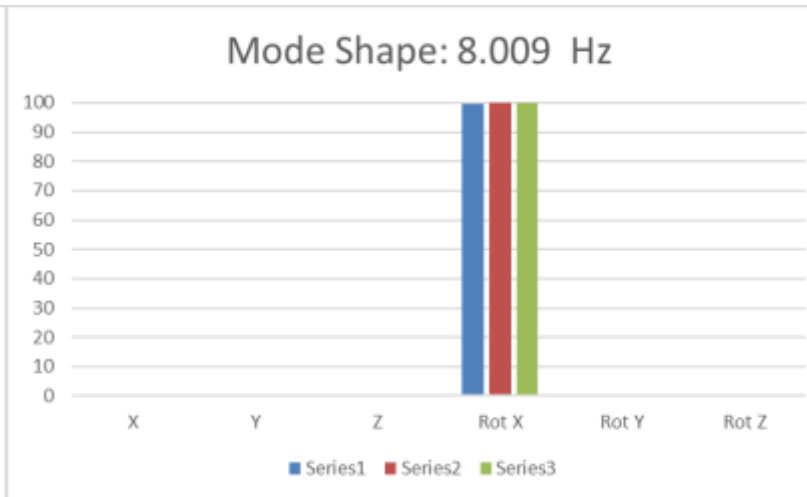
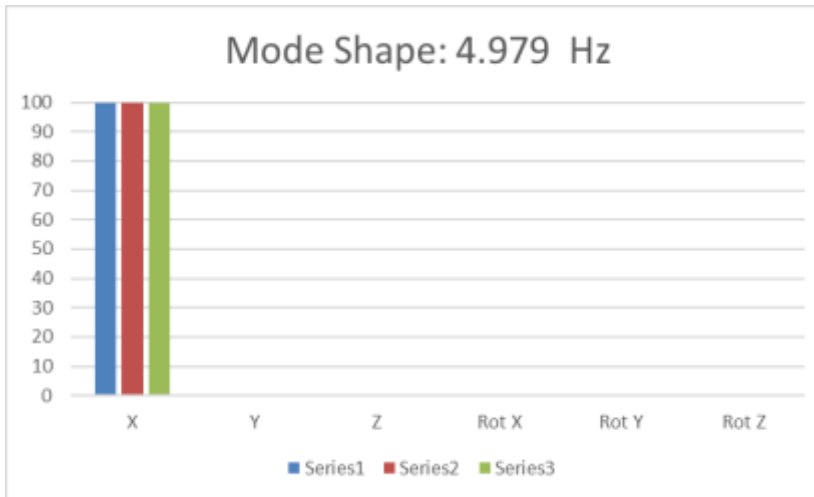
Eigen Frequency Correlation

Result from Internal Tool	4.979	8.009	11	16.18	17.808	20.937
Result From a Global Commercial Tool	4.9679	8.0124	10.9958	16.3297	17.808	20.9531
Result From COMSOL	4.979	8.009	11	16.18	17.808	20.937

Result From COMSOL

Modes (Hz)	4.979	8.009	11	16.18	17.808	20.937
X	99.999	0.205	0	0.378	0	0.007
Y	0	0.054	0	0.0027	99.987	5.31
Z	0	0	100	0	0	0
Rotation about X	0	99.723	0	0.166	0	1.195
Rotation about Y	0.001		0	99.151	0	0.095
Rotation about Z	0	0.017	0	0.278	0.012	93.392

Correlation: Mode Shapes



Series1: Result from Internal Tool

Series 2: Result From a Global Commercial Tool

Series3: Result From COMSOL

Summary

- The application calculates the modal frequencies and Kinetic energy distribution of a decoupled system accurately
- Helpful for designer in the early phase of development
- COMSOL App builder was used to prepare the customized applications (GUI) based on user requirements
- The web-based portal deployment via COMSOL server gives access to any of the users across different locations in the organization

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Thank You

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