Modeling the Hyperloop with COMSOL[®]: On the Mechanical Design of the EPFLoop Capsule

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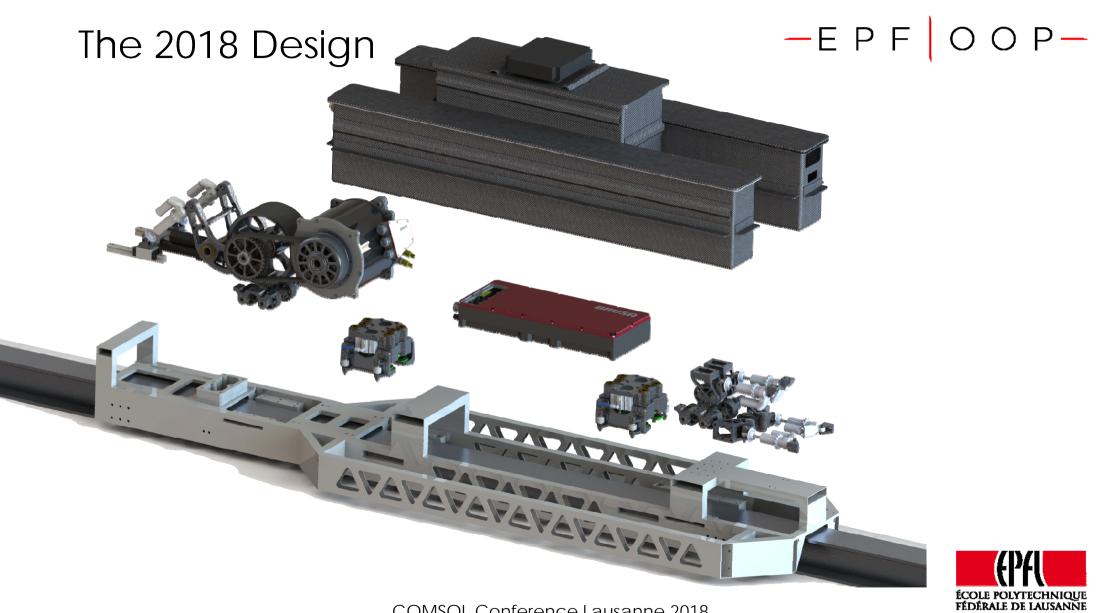
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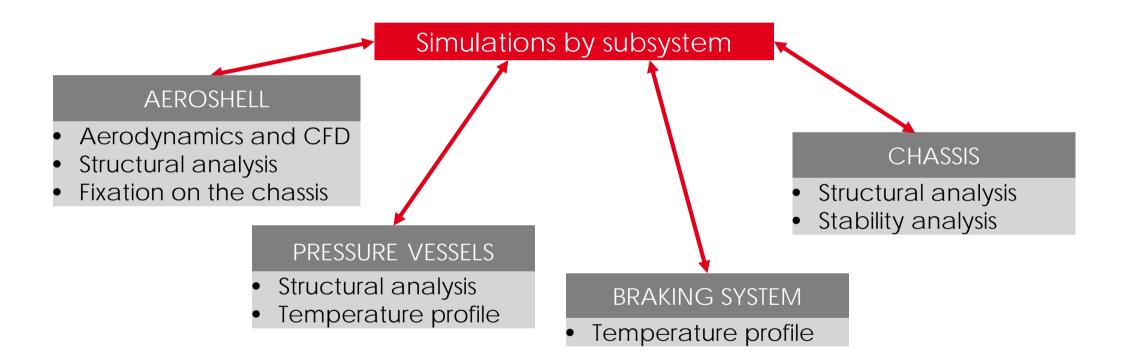
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THE 2018 POD DESIGN



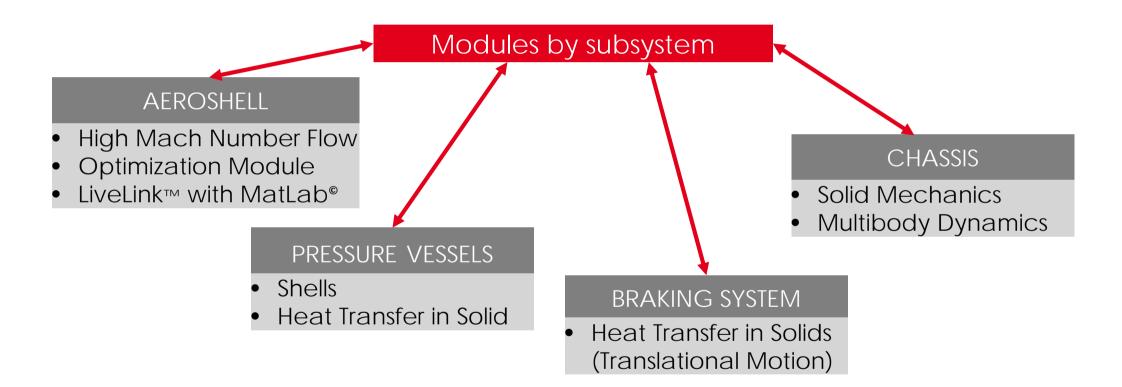


Subsystems Modeled with COMSOL – E P F | O O P –



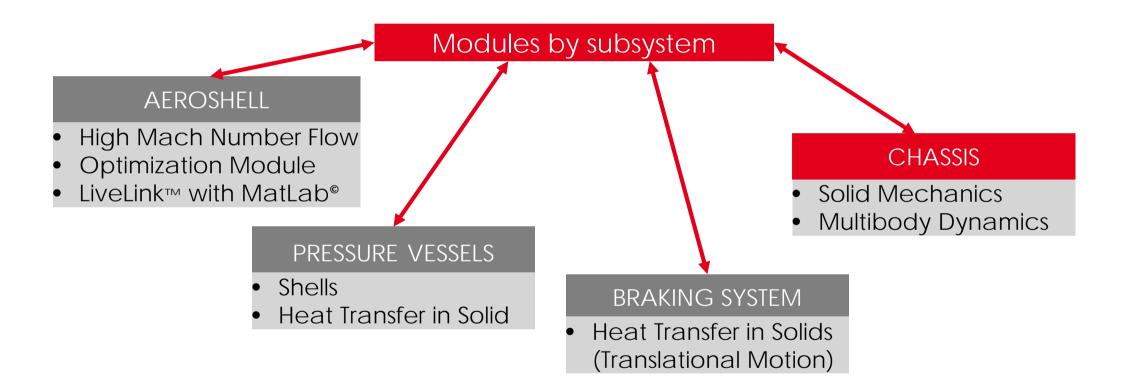


Subsystems Modeled with COMSOL – E P F O O P –





Subsystems Modeled with COMSOL – E P F O O P –

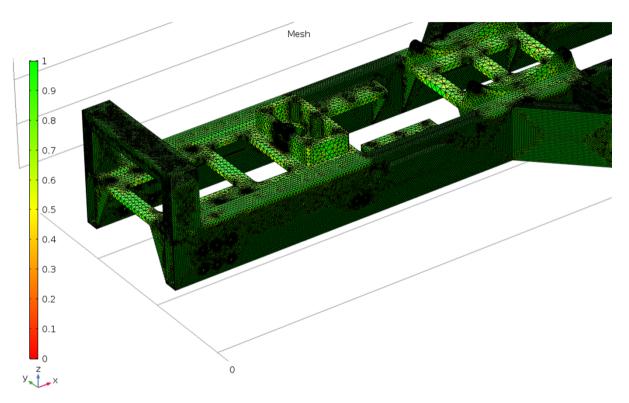




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Aim of the Simulations

- To evaluate the effect of the forces experienced during the run, a study of the stresses and the resulting displacements on the chassis has been done.
- In addition, due to the high speed and the possible occurrence of vibrations, the modelling of the stability system has been carried out with an eigenfrequency study.



Mesh on the chassis, mesh quality measured by skewness.



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Modules, Solvers and Strategies

Modules:

 Structural mechanics: The Solid Mechanics module has been used to simulate in stationary conditions the safety factor and the Von Mises stresses.

Strategies:

- The loads applied are acceleration (1.5 g), deceleration (3.0 g) and weight of the components.
- After a study using nominal loads, a load sweep and a linear buckling study have been performed to determine the maximum acceleration at which the chassis will withstand the stresses.

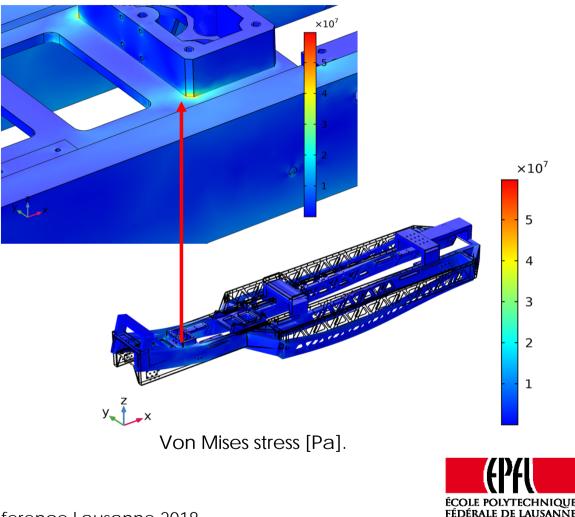
Type of Load	Fx [Acc]	Fx [Dec]	Fy [Gravity]
Motor	714 N	1428 N	500 N
VSI+Electronics+PV	336 N	672 N	235 N
Batteries	1204 N	2408 N	843 N
Stability System	154 N	308 N	107 N
Transmission	308 N	616 N	215 N
Cooling	70 N	140 N	49 N
Braking	224 N	448 N	156 N
Aeroshell	168 N	336 N	118 N



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Results

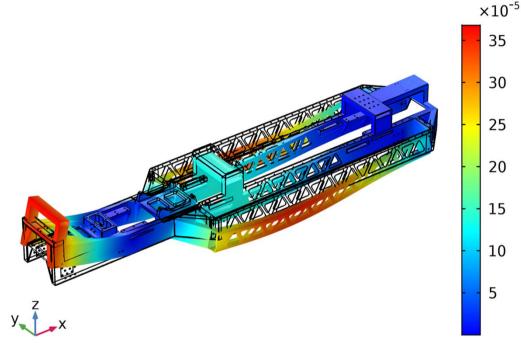
- In the structural analysis, the applied loads are the nominal loads given by each subsystems.
- The aluminum chassis (6060-T6) has an ultimate tensile strength of 275 MPa. Considering a safety factor of 2, the allowed octahedral stress is 65 MPa. The structural analysis showed that there are no stresses above this value on the structure.



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Results

- In the structural analysis, the applied loads are the nominal loads given by each subsystems.
- The aluminum chassis (6060-T6) has an ultimate tensile strength of 275 MPa. Considering a safety factor of 2, the allowed octahedral stress is 65 MPa. The structural analysis showed that there are no stresses above this value on the structure.
- The linear buckling study showed that the chassis can withstand the nominal loads up to 15 times with a safety factor of 2.



Total displacement [m].

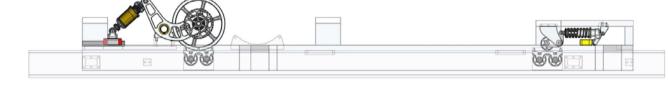


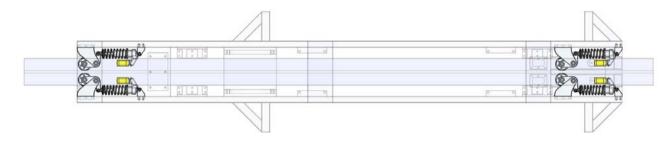
Simulations of the Stability System

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Aim of the Simulations

- The objective is to understand the behavior at high speed and sustained vibration of the pod.
- The stability system is composed by multiple subsystems:
 - Vertical suspensions (front and back), located over the rail
 - Set of wheels, located under the rail
 - Lateral suspensions (front and back) located between the chassis and the vertical part of the rail







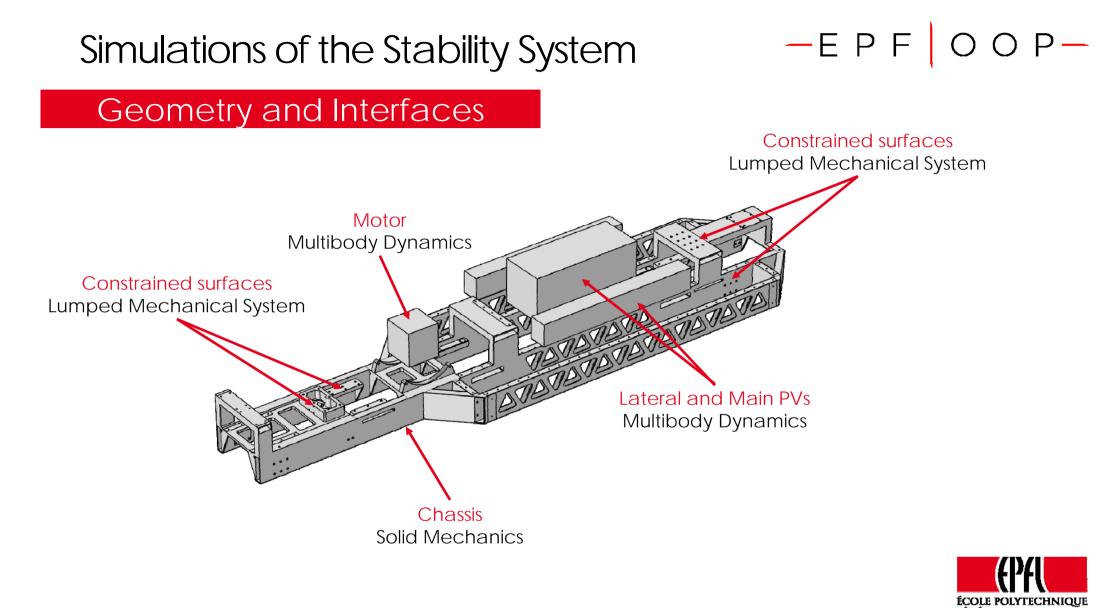
Simulations of the Stability System

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Modules, Solvers and Strategies

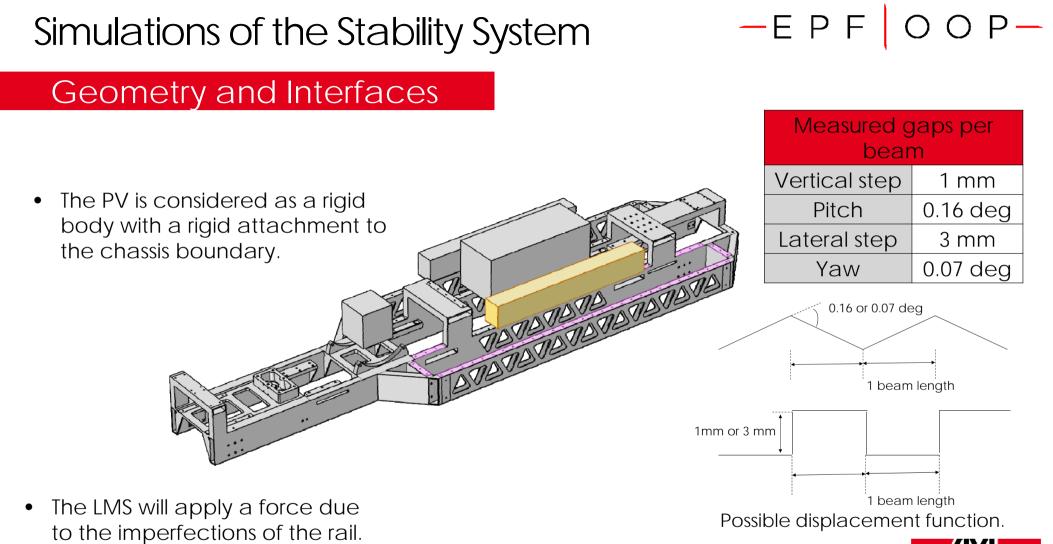
 Modules for the stability analysis: Multibody Dynamics, Lumped Mechanical Systems, Solid Mechanics have been used.





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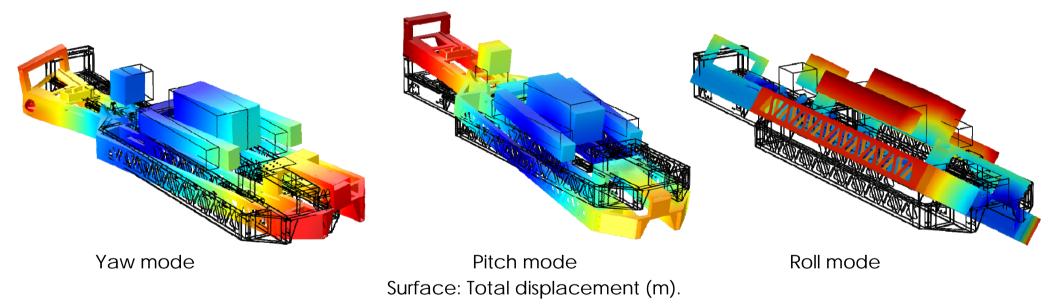
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Simulations of the Stability System

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Mode Analysis

• COMSOL provides as first three modes those that are rigid motions (and their complementary ones).



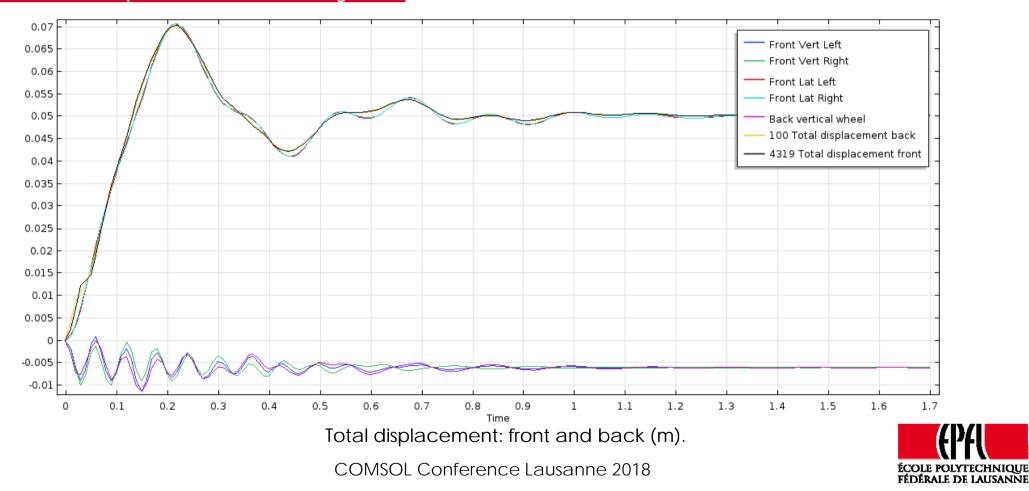
• The first natural mode of vibration has been computed at 50 Hz, whereas the induced vibrations have max frequency of 35 Hz.



Simulations of the Stability System

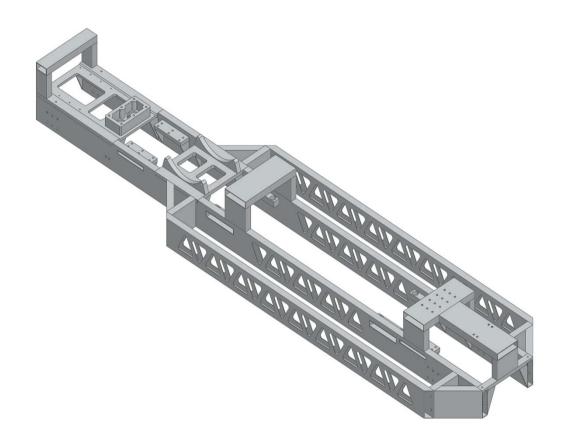
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Time Dependent Analysis



Manufacturing of the Chassis









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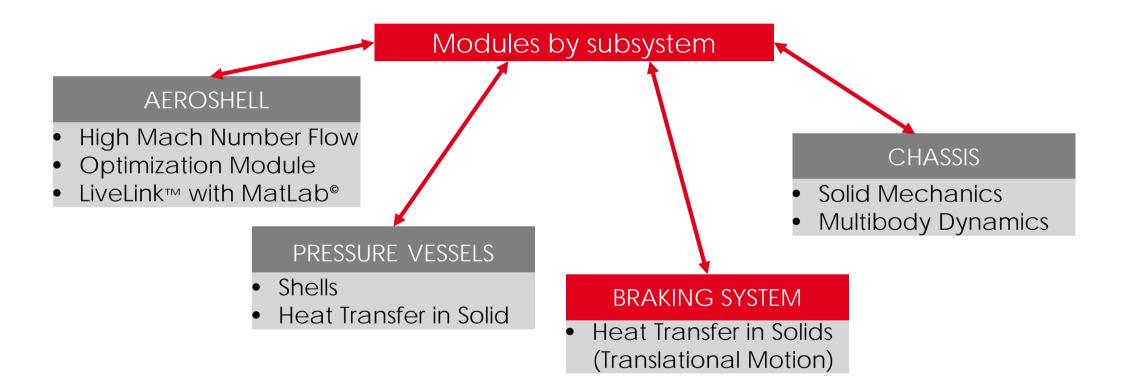


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Subsystems Modeled with COMSOL – E P F O O P –

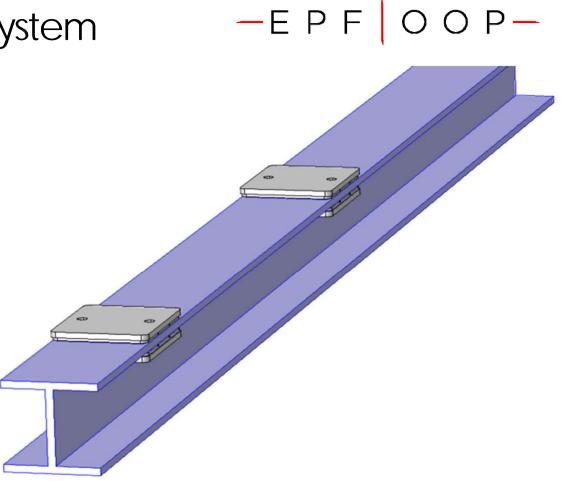




Simulations on the Braking System

Aim of the simulations

- In order to reach the top speed and then stop safely, it is required to have an efficient braking system.
- The amount of **kinetic energy** carried by the pod can create an excessive **increase of temperature** in the brakes.,
- Consequently, in order to choose correctly the material constituting the brakes and avoid to reach problematic temperatures, a set of simulations with heat transfer and frictional effects were performed.



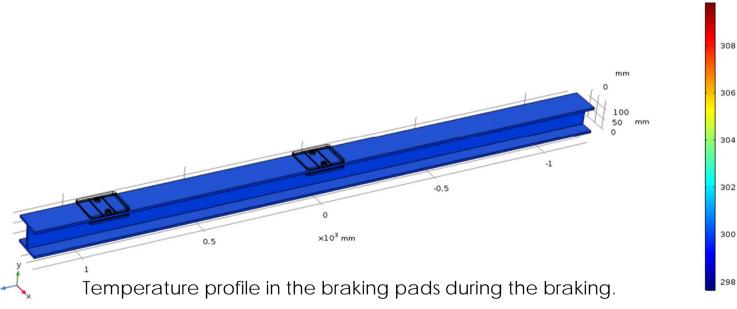


Simulations on the braking system

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Modules, Solvers and Strategies

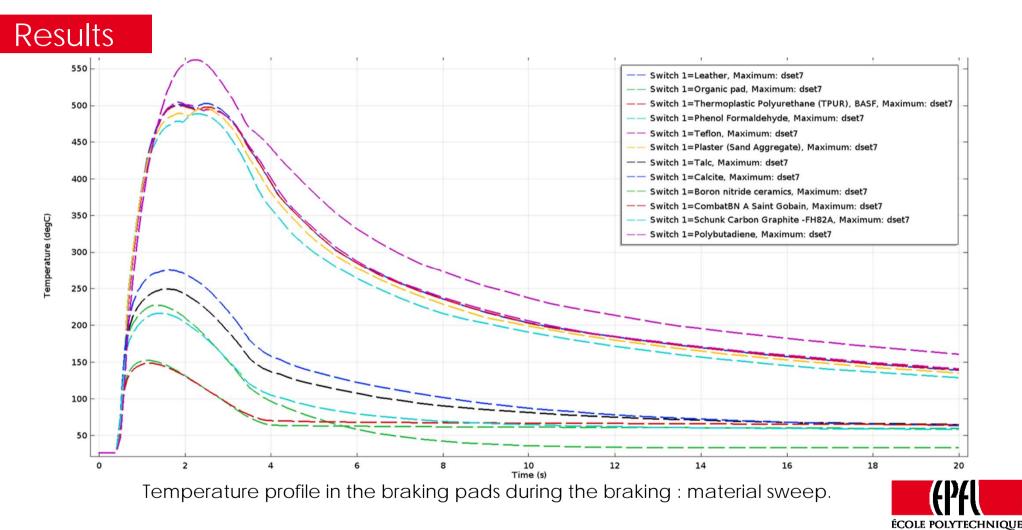
- Module: The Heat Transfer in Solid module has been used to simulate the temperature profile behavior of the brakes during the braking at the end of the run.
- Strategies: Using the Translation Motion and inserting the deceleration profile speed has been possible to estimate the power dissipated by friction on the rail and therefore the temperature rise in the pad volume.



Time=0 s Volume: Temperature (K)



Simulations on the Braking System

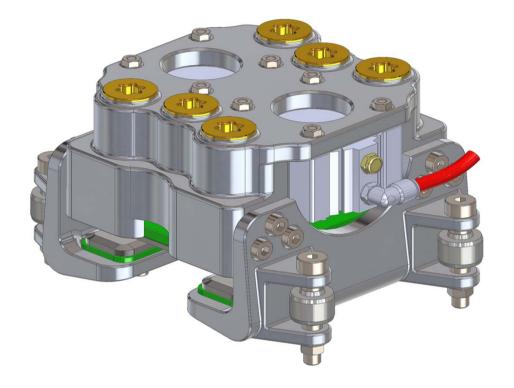


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Manufacturing of the Brakes

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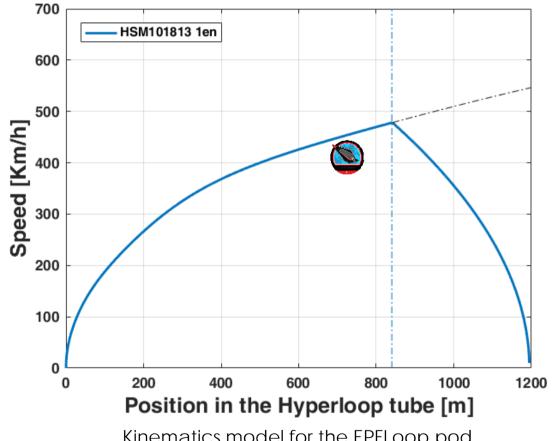
Simulations on the Braking System - E P F | O O P-

Validation The brakes have been successfully validated and the run celebrated.



Pod Performances Prediction

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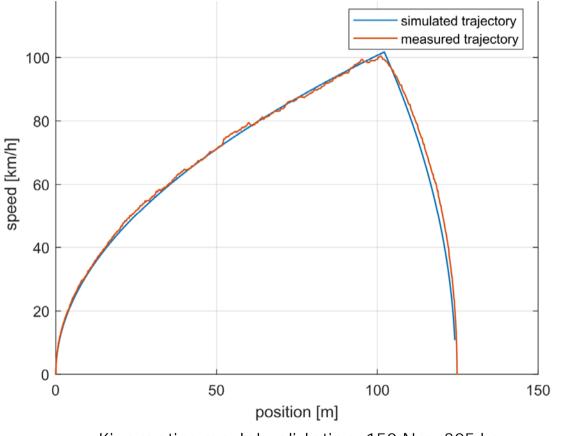
Mass	295 kg	
Max requested power	178 kW	
Max torque	385 Nm	
Total capacity	15 Ah	
Estimated max speed	470 km/h	

Kinematics model for the EPFLoop pod.



Pod Performances Validation

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 Comparison between the kinematics model developed by EPFLoop and measurements during the run

Kinematics model validation: 150 Nm, 305 kg.



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FUTURE: THE 2019 COMPETITION



The 2019 competition

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INCREASE MAXIMUM SPEED
SCALABLE PROTOTYPE
NEW APPROACH INSPIRED BY SWISSMETRO
OPTIMAL DESIGN
STRICT COLLABORATION WITH PARTNERS SUCH AS COMSOL
NEW INVOLVED STUDENTS



